

PATENT SPECIFICATION

Inventors: STANLEY VERNER SONMORI and ALFRED JAMES SAMWAYS

807.207



Date of Application and filing Complete Specification June 20, 1957.

No. 19468/57.

Complete Specification Published Jan. 7, 1959.

Index at acceptance: —Class 6(3), G6(B3: B4: J3: M: R).

International Classification: —A01d.

COMPLETE SPECIFICATION

Improvements in and relating to Lawn Mowers

We, TORO MANUFACTURING CORPORATION, a corporation organised according to the laws of the State of Minnesota, United States of America, of 3042 Snelling Avenue, Minneapolis, Minnesota, United States of America, do hereby declare the invention for which we pray that a patent may be granted to us, and the method by which it is to be performed, to be particularly described in and by the following statement:—

This invention relates to lawn mowers. Mowing machines, particularly power driven mowing machines, are generally quite heavy and as a result these machines are quite difficult to steer and have a distinct tendency toward leaving visible streaking effects on the lawn and sometimes permanent damage to the lawn when it is in a particularly soft condition. Such lawn mowers conventionally are provided with castored wheels at the front in order to permit proper steering. These castored wheels, however, are secured in such a manner and are so constructed and arranged that they have a very distinct tendency toward scuffing of the surface during turning operations and, because of the relatively long wheel base provided by them, the mower has a distinct scalping tendency. These machines are characterized by the difficulty which is experienced in steering them, especially on slopes or under such conditions that the weight of the machine tends to shift toward one of the castored wheels with the result that during turning operations and even when progressing in a straight line, there is a distinct tendency for the wheel bearing the greater weight to dig in and damage the lawn surface. The invention is designed to eliminate or reduce to a minimum these tendencies.

It is a general object of the invention to provide novel and improved means for supporting the forward end portion of a lawn mower which will obviate these drawbacks.

According to the present invention a lawn mower includes a frame supported at the rear

by wheels or like ground-engaging means, and having transversely spaced rotatable wheel-like members for supporting its front end, said members having convexly shaped ground-engaging surfaces and being mounted for rotation about axes extending upwardly and inwardly relative to said frame.

To enable the invention to be fully understood it will now be described with reference to the accompanying drawings, in which:—

Fig. 1 is a plan view of one embodiment of the invention;

Fig. 2 is a side elevational view of the same;

Fig. 3 is a vertical sectional view of one of the front supporting wheel members;

Fig. 4 is a diagrammatic view illustrating the rearward tilt of the axis of rotation of the frame supporting members; and

Fig. 5 is a diagrammatic view illustrating the inward tilt of the axis of the front supporting members.

As shown, in Figs. 1 and 2 of the accompanying drawings a lawn mower includes a frame indicated generally as F, the rearward portions of which are supported by a rear steering wheel 6 and a pair of intermediate wheels 7 and 8. The wheels 7 and 8 engage the ground and support the rearward portions of the frame while the forward portion of the frame such as 9, is supported by a pair of ground-engaging support members indicated generally as 10 and 11. As shown, these support members 10 and 11 are spaced transversely of the machine and are rotatably mounted on the forward end portions thereof.

The frame F includes an A-shaped structure indicated generally as 12, which has a cross plate 13 extending between the legs thereof. The legs of the frame 12 meet toward the rear. Rotatably mounted between the rear end portions of these legs, is a shaft 14 which extends vertically and carries a bifurcated member 15 at its lower end which in turn rotatably supports or mounts the wheel 6. Secured and keyed to the shaft 14 is a sprocket

wheel 16 which is driven by a chain drive 17.  
Mounted on the cross plate 13 and extending upwardly therefrom, is a steering shaft 18.  
This steering shaft is rotatably mounted on the plate 13 and is adapted to be turned through the use of the handle bars 19 and 20.  
Mounted on the shaft 18 and secured thereto for rotation therewith about the vertical axis of the shaft 18, is a gear member 21 which intermeshes with a gear 22 which is carried on a countershaft 23 which in turn is rotatably mounted on a bar 24. The shaft 23 also carries a sprocket 25 around which the chain drive 17 passes so that as a result of turning of the shaft 18, through the use of the handle bars 19 and 20, the gears 21 and 22 will cooperate to turn the shaft 23, and as a result, the chain drive 17 will cause the rear wheel 6 to be turned accordingly.  
Mounted on the frame F and illustrated only diagrammatically, is a motor and gear reducer indicated generally as M. The motor M is connected by a drive belt 26 which extends forwardly and passes around a pulley 27 to drive the same. The pulley 27 is mounted on a vertical shaft 28 which in turn is rotatably mounted upon a housing 29 which as best shown in Figure 2, has an open bottom and has a discharge outlet 30 at one of its sides. Mounted upon the lower end of shaft 28 is a cutter element or blade 31 which is secured to the shaft and rotates therewith to effect the cutting of the grass. The height of the cut can be adjusted by repositioning the washers or sleeves 32, which as shown in Fig. 2 are above the pulley 27 and are secured to the shaft by a nut 33. The elevation of the cutter element 31 can be adjusted by removing or adding an extra sleeve or washer 32 as desired. The pulley 27, of course, is drivably connected to the shaft 28 so that it will drive the cutter element 31 at relatively high speed within the housing 29 at an elevation slightly above the lower edge of the latter, as best shown in Fig. 2.  
The two rotatable support members 10 and 11 are constructed identically and therefore, for the sake of simplification, the parts of one of the two will be described with the understanding that the other is similar in construction. Each of the support members 10 and 11 is provided with a mounting bracket such as 34 which is secured to the vertical walls of the housing 29. Secured to each of these mounting brackets 34 is an inverted cup-shaped member 35, the structure of which can be best seen by reference to Fig. 3. This cup-shaped member is threaded internally at its lower end as at 36 to adapt the same to threadedly receive therein a bearing member 37. This bearing member 37, as best shown in Fig. 3, is externally threaded to engage the threads of the cup-shaped member 35 to rotatably mount therein a shaft 38. This shaft 38 is movable vertically in the bearing member 37, as can

best be seen in Fig. 3. Secured to the lower end of the shaft 38 is a disc member, indicated generally by the numeral 39. This disc or saucer-shaped member 39 has a relatively flat radius throughout most of its exterior surface as at 40, while the more peripheral portions thereof have a sharper radius as at 41. The relatively flat radius is quite important in attaining some of the advantages of the invention, as will be described in greater detail hereinafter, and as shown, is eight and one-fourth inches.  
Mounted within the cup-shaped member 35 is a spring member 42. This spring member 42 bears against the upper end of the shaft 38 and constantly urges it downwardly to the lower limits of its limited movement as is shown in Fig. 3. The upper end of the spring 42 is provided with a pressure plate 43 which is urged downwardly constantly by a pressure screw 44 which as shown in Fig. 3, is threaded into the upper end of the inverted cup-shaped member 35. A lock-nut 45 is provided for the pressure screw 44 to hold the same in the desired position. By varying the depth of the pressure screw 44, the amount of pressure which is applied to the spring 42 by way of the pressure plate 43 can be varied. It will be noted that the movement of the shaft 38 vertically has both upper and lower limits which are provided by a shoulder 46 in the cup-shaped member and a second shoulder 47 in the bearing member 37.  
Each of the rotatably mounted support members 10 and 11 is arranged so as to provide the advantages and obtain the objectives defined herein. The mounting or arrangement of these two support members can best be understood by reference to Fig. 4 and Fig. 5. Reference to Fig. 4 shows that each of these support members 10 and 11 are mounted in such a way on the outer surface of the housing, 29, that the axis of rotation thereof extends upwardly and rearwardly approximately 15° from vertical. Reference to Fig. 5 will show that each of the support members 10 and 11 are mounted so that the axis of rotation extends upwardly and inwardly about 20° from vertical. Thus these two support members are mounted so that their axes of rotation extend upwardly and rearwardly and inwardly toward each other.  
Each of the support members 10 and 11 is mounted for free rotation and they rotate only because they engage the ground and support the forward portions of the housing 29 and the frame F. It will be noted that because of their arrangement there will be a very slight amount of skidding which takes place when the machine moves directly forward in a straight line. However, the amount of skid which takes place is negligible and in general, the disc-shaped members 39 rotate so as to facilitate steering of the machine. The revolution of the disc-shaped members 39 is the pri-

mary movement and gives the effects and advantages of a swivel wheel. The effect of skidding on the turfs is negligible. The slight skidding referred to is the result of the inclination of the axis of the disc toward the center of the cutting unit.

In use, the weight of the machine is borne at the forward end thereof by the relatively broad, flat surfaces as at 40. This is the result of the inward and rearward tilting of the axis of the support members 10 and 11. The generous radius at 40 on each of the members, provides a broad weight-bearing surface which minimizes the damage to soft supporting surfaces such as a moist or relatively soft lawn. As a result of this broad weight-bearing surface, the adverse effect of the weight of a heavy machine, such as a power driven mower upon such a soft surface is minimized.

A further advantage of this broad weight-bearing surface 40 in contrast to the weight-bearing surface provided by a conventional castored wheel is that the tendency toward "streaking" is eliminated. With such a heavy machine, the conventional castored wheel presses the grass downwardly and flattens it to such an extent that the grass does not rise sufficiently before the cutter unit goes over it to do a proper job of cutting. As a result, after the grass has been cut, there will frequently be streaks across the cut surface which indicates where the conventional castor wheels have passed. By utilizing the broad weight-bearing surfaces 40, the weight is distributed over a wider area with the result that the grass springs up more quickly after the support members 10 and 11 have passed thereover, with the net result that a much improved cutting result is obtained.

The broad flat weight-bearing surfaces 40 have additional advantages when the machine is used on a slope. When a heavy machine, such as a power driven mower is utilized on a slope, the weight always shifts to the lower side. If the conventional castored wheel is utilized at the forward part of the machine, there is a distinct tendency because of this shifting of weight for the wheel to dig in and scuff or tear up the surface. This tendency is eliminated in the use of the support members 10 and 11, for the broad flat surface 40 spreads the weight over a wider area and is disposed substantially parallel to the surface because of the inward tilt of the axis of the support members, in addition to eliminating the digging and the tearing of the lawn surface, these support members also minimize the adverse effect of weight shifting downhill on the slope for its weight is distributed over a relatively broad area.

When a power driven lawn mower of the type shown, is provided with the rotatable support members 10 and 11, the result is that the tendency toward scuffing is minimized. This is especially true while negotiating turns and while cutting on a slope. "Scuffing" is a term utilized to describe the tendency for the wheel to slide over a lawn surface and severely damage the same without necessarily tearing the sod. It is the result of a member, such as a wheel, sliding sideways instead of moving forwardly in its normal line of motion.

The broad flat surfaces 40 provide other advantages in that they minimize the adverse effects of small depressions in the surface being traversed. For example, a small depression may readily pass beneath the broad, relatively flat surface 40 without any indication thereof being seen in the tilt or movement of the machine. This is in sharp contrast to what would be experienced if a castored wheel of the conventional type were utilized. As a result, the distinct scalping tendency which results from a conventional castored wheel when it enters such a depression is eliminated for the machine is supported at the same level even though such a depression is passed over.

It should also be noted that through the use of the relatively broad flat surfaces 40 and the angle of inclination of the axis of the support members 10 and 11, the adverse effect of weight shifting to the outside during turning has been minimized. In turning a machine provided with conventional castored wheels, the weight of the machine which always shifts to the outside during a turn, causes such wheels to dig in and damage the surface. Despite the shifting of weight in the use of my machine, the broad flat surfaces 40 preclude and substantially eliminate the adverse effects normally associated therewith.

It will be noted by reference to Fig. 1 that the support members 10 and 11 are secured to the housing 29 in such a manner that a substantial portion of the disc-shaped members extends beneath the housing 29 and also within the orbit of the cutter element 31. This arrangement is important for it provides a substantially shorter wheel base for the machine. Reference to Fig. 1 shows that it is possible to support the forward end of such a machine with the support members 10 and 11 without materially adding to the wheel base of the machine. Anything which tends to shorten the wheel base of such a machine, tends to reduce the tendency toward "scalping". By positioning the disc-shaped members 39 as shown in Fig. 1, we have reduced the wheel base of such a machine to a minimum, and thus there is a substantially less tendency toward "scalping" when using our machine.

It will also be noted that the disc-shaped members 39 are so mounted that the outer periphery thereof extends outwardly only slightly farther than the more lateral portions of the housing 29 at the right hand side of the machine. This positioning of the saucer-shaped member 39 enables it to be utilized as a guide when the machine is used to trim along a wall or the like. When a relatively

heavy machine such as a power driven rotary mower as is shown in Figs. 1 and 2, is utilized on the side of a hill, the downward shifting of the weight of such a machine, makes steering thereof difficult. We have been able to successfully counteract this tendency by mounting the support members 10 and 11 so that the axis of rotation of the disc-like members 39 extend inwardly. As a result of this orientation of the axis of rotation of the saucer-shaped members 39, there is a distinct tendency for the members 39 to turn inwardly and as a result, this angle of the disc or saucer-shaped members 39, counteracts the downward shifting of the weight of the machine. In other words, the inclination of the axis of rotation of the saucer-shaped members 39 provides a distinct tendency for the machine to crawl upwardly relative to the slope, with the result that the adverse effects normally experienced by the downward shifting of weight on the side of a hill is counteracted and steering becomes a relatively easy matter despite the fact that the machine is being used on a sharp slope.

The saucer-shaped members act to facilitate steering in a similar manner while a turn is being negotiated. The outward shifting of weight of the machine on such a turn, makes it difficult normally to turn the machine, but here again the angle of the axis of rotation of the saucer-shaped members 39 is such that the member 39 tends to turn the machine in the direction that the operator is attempting to turn the machine. Thus, the weight always shifts to the outside. In our machine it shifts upon the saucer-shaped member which tends to turn the machine in the desired direction and the weight is shifted off the other saucer-shaped member 39 which would tend to turn the machine in the opposite direction. In other words, the shifting of the weight is utilized to benefit the operator by aiding him in turning the machine.

It will be noted that we have shortened the wheel base of our machine more than the radius of the conventional castored wheel which is utilized at the front of such a machine conventionally. Because of the shortened wheel base, our machine can be steered more easily, for it is well known and recognized that the shorter the wheel base of a machine the more easily it may be turned, everything else being equal.

The positioning and mounting of the saucer-shaped members 39 so that they pass beneath the lower peripheral edge of the housing 29 and also below the cutter element 31 provides a number of distinct advantages. The tendency toward "scalping" by both the blade and the lower edge of the housing 29, is substantially reduced and practically eliminated because of the shortened wheel base which as previously explained, tends to eliminate "scalping". Also the disc-shaped member 39

acts as a guard for the blade and since it extends beneath the housing, it also acts as a positive guard member to preclude the housing digging into the sodded surface when sharp undulations in the terrain are encountered.

The spring-loading of the saucer-shaped members 39 is also important for it eliminates the shifting of weight of the machine and jiggling (i.e. lateral to and fro movement as well as up and down movement) and deflections thereof which are normally caused by the encountering of small depressions with the conventional wheels. When such a small depression is encountered, by the saucer-shaped members 39, there is no adverse effect and no tendency for the weight of the machine to shift and no jiggling or deflections of the machine, for the relatively broad surfaces 40 easily pass thereover with no adverse effect upon the machine. If the depression is sufficiently large so that the saucer-shaped member extends downwardly into it, the spring-loading takes up the vertical space provided by the depression, for the shaft 38 moves downwardly and again there is no shifting of weight, jiggling or deflection. In other words, the spring members 42 compensate for variations in the contour of the terrain. As a result, there is a pronounced stabilizing effect upon steering and there is no tilting or jiggling of the machine caused by such depressions.

Another advantage of our machine is that the variations in cut and the tendency to scalp on turns has been substantially eliminated. If conventional castored wheels are utilized, the variation in cut is provided substantially by adjustment of these castored wheels. This means that the machine rotates about the axis of the rear supporting wheels and as a result, while negotiating a turn there is frequently experienced a sharp variation in the height of the cut. These variations in cuts at the turn are not experienced with our machine.

The relatively sharp radius at the more peripheral portions as at 41, on the disc-shaped members 39, brings the leading edge of the disc upward to act as a skid or deflector in case relatively large objects are encountered and facilitates climbing over such obstacles.

From the above, it can be seen that the use of the saucer-shaped members 39, constructed and arranged as shown and described, provide a machine which gives equal maneuverability as compared to a machine having a supporting front wheel of relatively large diameter and yet provides many advantages which cannot be obtained with a wheel of such relatively large diameter. These support members 10 and 11 enables the mower to negotiate severe ground undulations without scalping and other related adverse effects. The compactness of our machine gives a mower with good side trimming characteristics and yet keeps the scalping characteristics within reasonable limits.

It will be noted that the discs 39 are spring-

807,207

5

5 loaded with a spring of sufficient strength so  
that when the load of the machine is divided  
equally between the two discs 39, they do not  
compress if in operation a small bump is  
15 encountered by one of the discs. In that event,  
more load is applied to the disc encountering  
the bump, and it moves in an upward direction  
by compressing the spring. Thus the use of  
10 the springs produce a pronounced stabilizing  
effect in steering whether a bump or a depres-  
sion is encountered.

WHAT WE CLAIM IS:—

15 1. A lawn mower including a frame sup-  
ported at the rear by wheels or like ground-  
engaging means, and having transversely  
spaced rotatable wheel like members for sup-  
porting its front end, said members having  
convexly shaped ground-engaging surfaces  
20 and being mounted for rotation about axes  
extending upwardly and inwardly relative to  
said frame.

2. A lawn mower according to Claim 1  
wherein the axes of said members also extend  
upwardly and rearwardly.

25 3. A lawn mower according to either of the  
preceding claims wherein said front support  
members are of saucer-like shape having con-  
vexly shaped surfaces for engaging the ground  
as they rotate.

30 4. A lawn mower according to Claim 3,  
wherein the radius of curvature of said ground-  
engaging surfaces exceeds seven inches.

5. A lawn mower according to any one of  
the preceding claims wherein the axes of rota-  
35 tion of said front support members extend  
about 20° upwardly and inwardly from the  
vertical.

6. A lawn mower according to any one of  
the preceding claims wherein the axes of rota-

tion of said front support members extend 40  
upwardly and rearwardly about 15° from the  
vertical.

7. A lawn mower according to any one of  
the preceding claims wherein at least one of  
said front support members is so arranged and 45  
disposed that the periphery of said support  
member extends only slightly laterally out-  
wardly of the confines of the machine so as to  
function as a guide when the machine is moved  
along a wall or the like. 50

8. A lawn mower according to any one of  
the preceding claims wherein each front sup-  
port member is resiliently supported by means  
permitting limited movement in an upward  
direction. 55

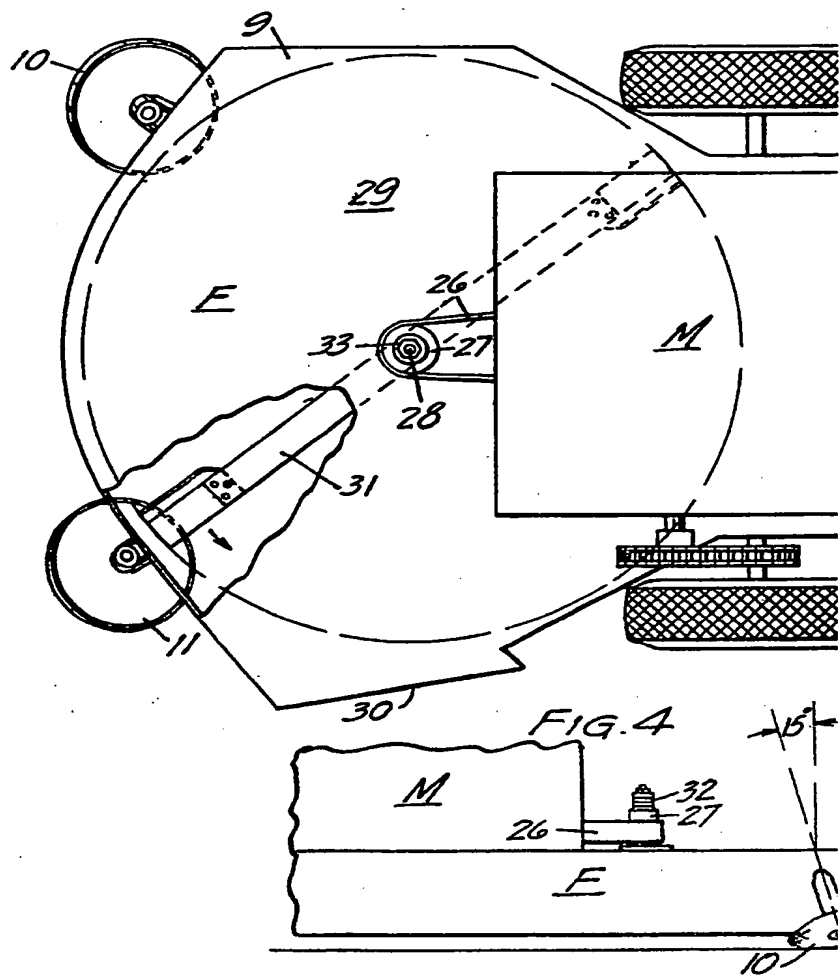
9. A lawn mower according to any one of the  
preceding claims including a housing having  
an open bottom and depending wall structure,  
a rotatable cutter element mounted for rotation  
within said housing and rotating in close prox- 60  
imity to the depending wall structure.

10. A lawn mower according to Claim 9  
wherein parts of said front support members  
extend beneath the confines of said housing  
and beneath and within the circle of rotation 65  
of said cutter element.

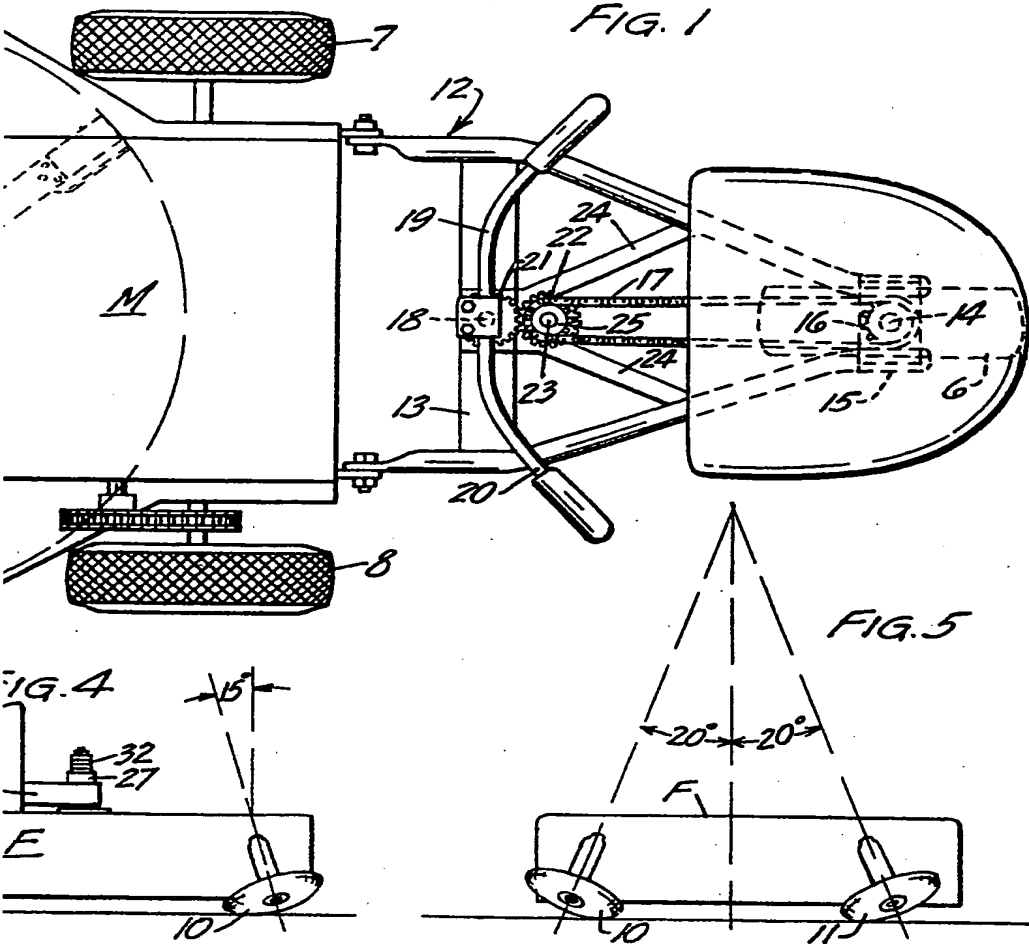
11. A lawn mower according to Claim 10  
wherein the axes of rotation of said members  
lie immediately outside and forward of said  
housing. 70

12. A lawn mower substantially as described  
and illustrated with reference to the accom-  
panying drawings.

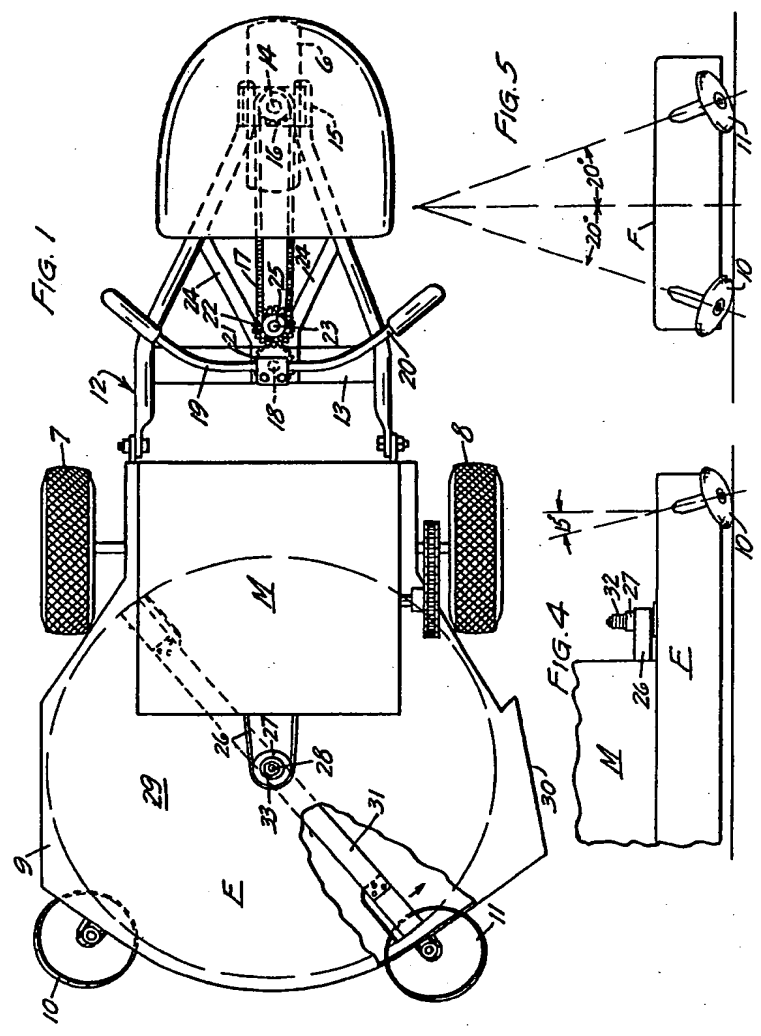
ABEL & IMRAY,  
Agents for the Applicants,  
Quality House, Quality Court,  
Chancery Lane, London, W.C.2.



807,207 COMPLETE SPECIFICATION  
2 SHEETS  
This drawing is a reproduction of  
the Original on a reduced scale.  
SHEET 1



807,207 COMPLETE SPECIFICATION  
2 SHEETS  
This drawing is a reproduction of  
the Original on a reduced scale.  
SHEET 1





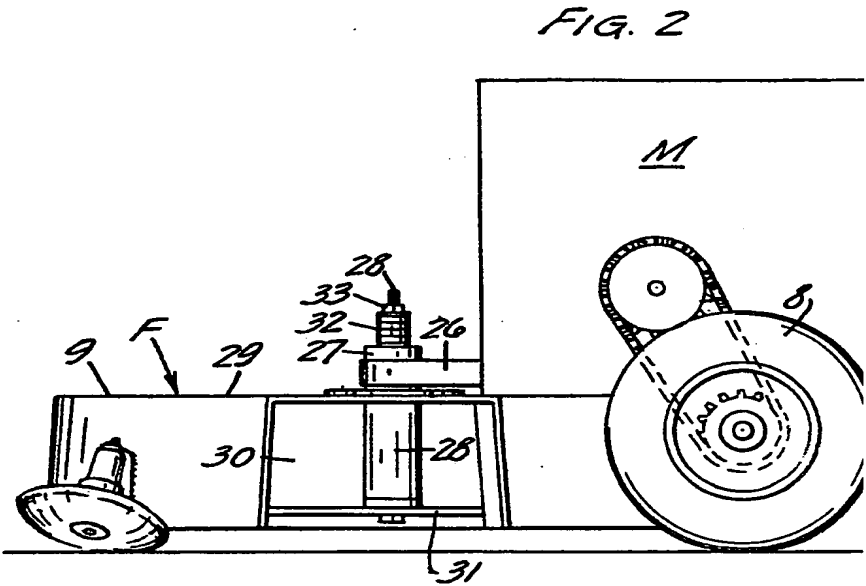
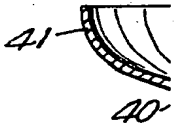
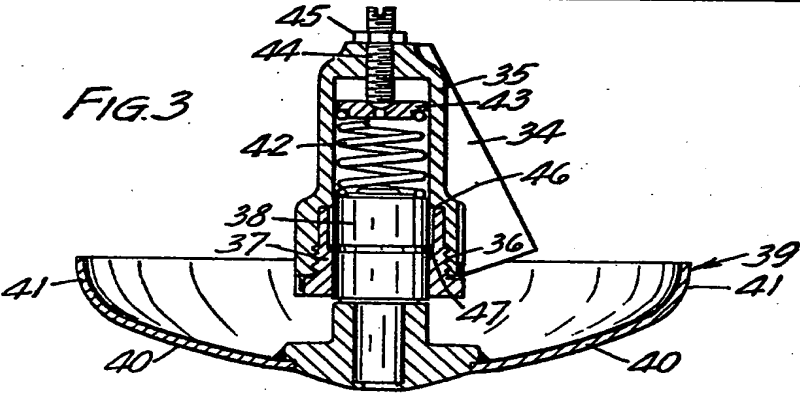
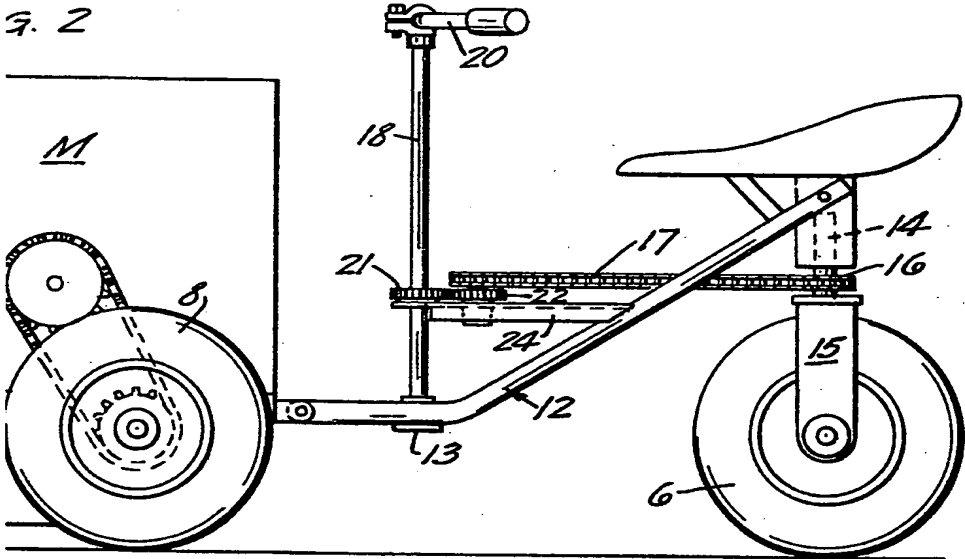


FIG.-



807,207 COMPLETE SPECIFICATION  
2 SHEETS  
This drawing is a reproduction of  
the Original on a reduced scale.  
SHEET 2



807,207 COMPLETE SPECIFICATION  
2 SHEETS  
This drawing is a reproduction of  
the Original on a reduced scale.  
SHEET 2

